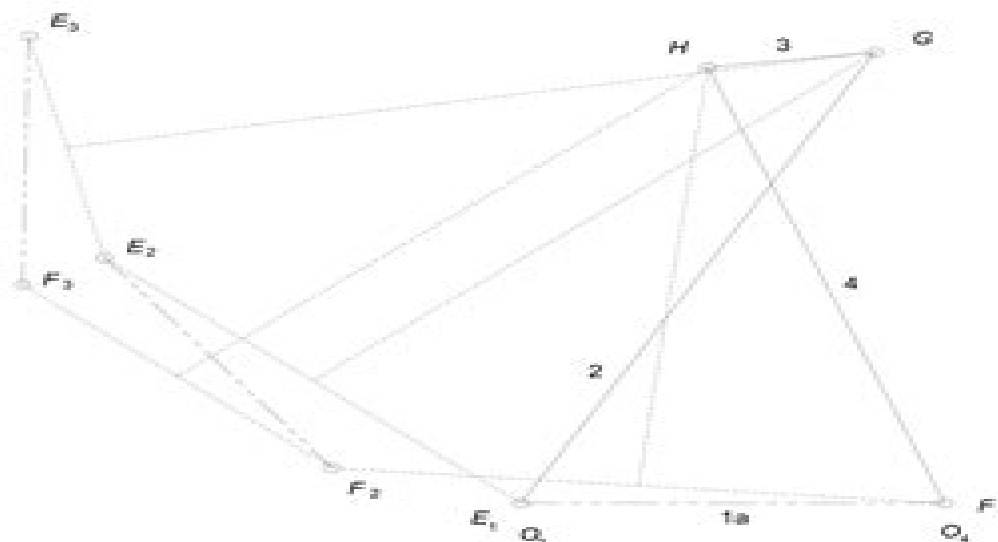


9. Bisect line E_1E_2 and line E_2E_3 and extend their perpendicular bisectors until they intersect. Label their intersection G .
10. Repeat steps 2 and 3 for lines F_1F_2 and F_2F_3 . Label the intersection H .
11. Connect E_1 with G and label it link 2. Connect F_1 with H and label it link 4. Reverting, E_1 and F_1 are the original fixed pivots O_2 and O_4 , respectively.
12. Line GH is link 3. Line O_2O_4 is link 1a (ground link for the fourbar). The fourbar is now defined as O_2GHQ_4 and has link lengths of

Ground link 1a	$L_{1a} = 4.303$	Link 2	$L_2 = 8.597$
Link 3	$L_3 = 1.711$	Link 4	$L_4 = 7.921$



13. Check the Grashof condition. Note that any Grashof condition is potentially acceptable in this case.

```

Condition( $a, b, c, d$ ) :=
  S ← min( $a, b, c, d$ )
  L ← max( $a, b, c, d$ )
  SL ← S + L
  PQ ←  $a + b + c + d - SL$ 
  return "Grashof"   if SL < PQ
  return "Special Grashof" if SL = PQ
  return "non-Grashof" otherwise

Condition( $L_{1a}, L_2, L_3, L_4$ ) = "Grashof"

```

The fourbar that will provide the desired motion is now defined as a Grashof double crank in the crossed configuration. It now remains to add the original points C_1 and D_1 to the coupler GH and to define the driving dyad.

Design Of Mechanism Norton 5th Solution Manual

Lauren Gardner



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